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(54) METHOD OF QUICK-OPENING A SET OF ROLLS IN A CALENDER, IN PARTICULAR IN A SUPERCALENDER, AND A HYDRAULIC SYSTEM FOR A SET OF ROLLS IN A CALENDER, IN PARTICULAR A SUPERCALENDER

VERFAHREN ZUR SCHNELLÖFFNUNG EINES WALZENSTAPELS IN EINEM KALENDER,
INSBESONDERE EINEM SUPERKALENDER UND HYDRAULISCHES SYSTEM FÜR EINEN
WALZENSTAPEL IN EINEM KALENDER, INSBESONDERE EINEM SUPERKALENDER

PROCEDE POUR L'OUVERTURE RAPIDE DES ROULEAUX DE L'ENSEMBLE DES ROULEAUX
D'UNE CALANDRE, EN PARTICULIER UNE SUPERCALANDRE, ET SYSTEME HYDRAULIQUE
POUR UN ENSEMBLE DE ROULEAUX D'UNE CALANDRE, EN PARTICULIER UNE
SUPERCALANDRE

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Description

[0001] The invention relates to a method of quick-opening a set of rolls in a calender, in particular a supercalender.

[0002] The invention also relates to a calender, in particular a supercalender, having a set of rolls comprising a variable-crown top roll, a variable-crown bottom roll and roll two or more intermediate rolls fitted between the top roll and the bottom roll, which rolls are arranged to form a substantially vertical stack of rolls, the rolls placed one above the other in the set of rolls being in nip contact with one another and forming calendering nips therebetween, in which set of rolls the intermediate rolls are provided with hydraulic relief cylinders, so as to relieve the nip load produced by the masses of bearing housings of the intermediate rolls and of auxiliary devices associated therewith, such as, take-out rolls, the calender comprising a hydraulic system for the set of rolls which is arranged to supply hydraulic oil to the relief cylinders under a desired pressure during operation of the calender, which pressure depends on the load acting on the relief cylinders.

[0003] The set of rolls in a supercalender conventionally comprises a plurality of rolls, which are arranged one above the other as a stack of rolls. The rolls placed one above the other are in nip contact with one another, and the paper or board web or equivalent to be calendered is arranged to run through the nips between the rolls. The rolls are journaled revolvingly on bearing housings, which in turn are normally attached to base parts fitted slidably on vertical guides provided in the frame of the calender. The base parts are suspended through stop parts on vertical lifting spindles provided in the frame of the calender, one of the functions of said lifting spindles thus being to act as guides to keep the rolls of the set of rolls in a correct position. Thus, the rolls of the set of rolls are not rigidly fixed at their bearing housings to the frame of the calender, but, instead, the rolls can move in a vertical direction.

[0004] The masses of the bearing housings of the rolls and of the auxiliary devices attached thereto, such as, take-out rolls, are quite large, which caused in conventional supercalenders the considerable drawback that said masses caused distortions in the distributions of the linear loads of the nips. For this reason, supercalenders started to employ relief devices, which are supported on the base parts of the rolls, on one hand, and on spindle nuts provided on the lifting spindles, on the other hand, so that the distortions caused by the weight of the bearing housings of the rolls and of the auxiliary devices attached thereto in the linear load profiles between the rolls were relieved by means of relief devices. One such arrangement has been previously disclosed, inter alia, in *US Patent 4,901,637*. The use of relief devices is previously known also from conventional machine calenders, in which attempts are made to eliminate, in particular, by means of hydraulic relief cylinders the above-

mentioned effect of concentrated loads arising from the bearing housings of the rolls and from auxiliary devices.

[0005] In conventional sets of rolls of supercalenders provided with a lifting spindle, in connection with the quick-opening of the set of rolls, the rolls fall so as to rest on a nut of the lifting spindle situated below a bracket of the base part of the bearing housing. The quick-opening of the set of rolls has been carried out, as a rule, in such a way that the pressure has been discharged out from the lifting cylinder of the bottom roll in the set of rolls or, in a similar way, in the case where as the bottom roll is used a roll which is adjustable in zones and in which the roll mantle is able to move over its entire length in a radial direction relative to the axle of the roll, the pressures have been discharged out from the loading members of the roll mantle so that the roll mantle has substantially moved relative to the axle of the roll, thereby allowing the intermediate rolls of the set of rolls to "fall" a certain distance downwards together with the roll mantle of the bottom roll. The movement of the lower intermediate rolls has been larger than the movement of the upper intermediate rolls in order to provide a gap of substantially equal size between the rolls. For this reason, the opening of the nips of the set of rolls has not been simultaneous, but, instead, the opening of the set of rolls has started from the uppermost nip, proceeding as a successive process towards the lowermost nip. This has usually taken place irrespective of whether or not any relief devices have been used in connection with the intermediate rolls. Additionally, the opening distances have been previously quite considerable because supercalenders previously employed fibre rolls, or so-called filled rolls, as soft rolls. The opening of the nips at different times has caused the considerable disadvantage that there have been substantial variations in the tension of the web and, consequently, there have always occurred web breaks in quick-opening. The broken end of paper has in turn run through the closed nips, which has resulted in the marking of soft rolls, which has further caused that the quality of paper has been completely spoilt, unless said marked roll has been replaced.

[0006] New supercalenders have begun to employ polymer rolls as soft rolls instead of fibre rolls, whereupon the total height variation of the set of rolls has remained considerably smaller than in conventional supercalenders that use filled rolls. One reason for this reduced total height variation has been the fact that variations in the diameters of soft rolls have been very small because the grinding allowances of said rolls are small. This reduced total height variation has in turn led to the fact that it has been possible to omit altogether the lifting spindles and the glide guides associated with conventional sets of rolls in a supercalender, and it has been possible to replace this construction with a so-called articulated set of rolls, where the intermediate rolls of the set of rolls are mounted on the frame of the calender by means of articulated arms pivotally mounted on said

frame. It has been possible to do so for the reason that, since the variations of the total height of the set of rolls are small, the vertical distance of movement required by an individual roll is small, and because of this, irrespective of the articulated attachment mode, the movements of the rolls relative to the nip level have also been very small when adjusting the height position of the roll.

[0007] One such very developed articulated set of rolls is disclosed, *inter alia*, in *FI Patent Application No. 935214*. However, the problem also with this kind of supercalender has been to make the nips of the set of rolls open simultaneously during quick-opening of the set of rolls. For this reason, the purpose of the present invention is to provide a method and a system by which the nips of the set of rolls can be opened in a controlled and a desired manner in the order desired during quick-opening, preferably such that the nips open simultaneously during quick-opening.

[0008] With a view to attaining the objectives of the invention, the invention provides a method of quick-opening a set of rolls in a calender, in particular a supercalender, in which calender a paper or board web to be calendered is passed through calendering nips formed by a variable-crown top roll, a variable-crown bottom roll and by two or more intermediate rolls fitted between the top and bottom rolls, which rolls are arranged to form a substantially vertical stack of rolls; and in which calender the intermediate rolls are supported by means of hydraulic relief cylinders so as to relieve the nip load produced by the masses of bearing housings of said intermediate rolls and of auxiliary devices associated therewith, whereby during the quick-opening of the set of rolls, the bottom roll of the set of rolls is lowered and the relief pressures in the hydraulic relief cylinders are discharged completely or partly so as to open the calendering nips, and whereby the hydraulic oil flowing out from the relief cylinders is controlled by means of valves connected to the relief cylinders of each intermediate roll so that, during quick-opening of the set of rolls, the roll nips may be opened in a controlled way in a desired order.

[0009] The calender according to the invention is mainly characterized in that the hydraulic system is arranged to supply hydraulic oil to the relief cylinders through valves and that, during the quick-opening of the set of rolls when the bottom roll in the stack of rolls has been lowered, hydraulic pressure is discharged from the relief cylinders in a controlled way by means of said valves so that the calendering nips may be opened in a desired order.

[0010] The invention provides significant advantages over prior art arrangements and of these advantages, among other things, the following may be stated. The quick-opening operation has been accomplished in the invention most advantageously completely hydraulically so that the nips may be opened in a desired manner and quickly. In the most advantageous embodiment the invention is accomplished such that the nips open simul-

taneously. For instance, in a web break situation the simultaneous and quick opening of the nips is of crucial importance, because in supercalenders the quick-opening should, as a rule, take place even in a shorter time than half a second after a web break has been observed. This can be accomplished by the arrangement of the invention. The further advantages and characteristic features of the invention come out from the following detailed description of the invention.

[0011] In the following, the invention will be described by way of example with reference to the figures in the accompanying drawing.

[0012] Fig. 1 is a schematic side view of a calender to which the method and the system in accordance with the invention are applied.

[0013] Fig. 2 shows a hydraulic circuit diagram of the system in accordance with the invention in a situation where the calendering nips have been opened.

[0014] Fig. 2A shows by means of a circuit diagram corresponding to Fig. 2 a calender in a running position with the calendering nips closed.

[0015] Figs. 3A—3C, 4A—4C and 5A—5C show some alternative cylinder applications, which are suitable for use in the hydraulic system in accordance with the invention in relief cylinders of calender rolls.

[0016] Thus, Fig. 1 is a schematic side view of a supercalender in which the method and the system in accordance with the invention are applied. The supercalender in Fig. 1 is denoted generally with the reference numeral 10, and it comprises a calender frame 11, in which a stack of rolls 12 consisting of a number of rolls has been mounted in a vertical plane. The stack of rolls 12 comprises a top roll 13, a bottom roll 14, and a number of intermediate rolls 15—22 fitted one above the other the top roll and the bottom roll, which rolls are all arranged in such a way that they are in nip contact with one another in a running situation. Fig. 1 shows the stack of rolls 12 of the calender expressly in the running position. A paper or board web W is passed in the example shown in Fig. 1 over a spreader roll 133 and a take-out roll 134 into an upper nip N₁ and further through other nips N₂—N₈ in the calender and finally out from a lower nip N₉. Between the nips N₁—N₉ the paper or board web W is taken apart from the face of the rolls by means of take-out rolls 154, 164, 174, 184, 194, 204, 214, 224. As shown in Fig. 1, the take-out rolls are situated alternately "outside" (take-out rolls 164, 184, 204, 224) and "inside" (take-out rolls 154, 174, 194, 214) the stack of rolls relative to the calender frame 11.

[0017] The top roll 13 in the calender is a variable-crown roll, whose bearing housing 131 in the embodiment of Fig. 1 is attached directly and rigidly to the calender frame 11. The axle of the variable-crown top roll 13 is mounted in said bearing housing 131, and the roll is provided, in a normal way, with internal loading means, by which the deflection of the roll mantle may be regulated in a desired way. The top roll 13 is a so-

called soft roll and it is provided with a resilient polymer coating 132.

[0018] In a similar way, the bottom roll 14 in the calender is a variable-crown roll, whose roll mantle is mounted rotatably on the roll axle and which roll 14 is provided with internal loading means, by which the deflection of the roll mantle can be regulated in a desired way. The axle of the bottom roll 14 is mounted in bearing housings 141, which in the embodiment of Fig. 1 are mounted on loading arms 144, which are linked by means of articulated joints 145 to the calender frame 11. Between the calender frame 11 and the loading arms 144 there are mounted lower cylinders 143, by whose means the bottom roll 14 can be displaced in a vertical plane. Thus, the loading of the nips N_1 — N_9 in the stack of rolls 12 can be provided by means of the lower cylinders 143 and, further, the stack of rolls 12 can be opened, if needed, by means of said lower cylinders 143. The internal loading means of the top roll 13 may also participate in the loading of the nips N_1 — N_9 in the stack of rolls. Owing to the variable-crown bottom roll 14, the profiles of linear loads can be kept uniform in the nips N_1 — N_9 in the stack of rolls 12. In the example of the figure the bottom roll 14 is also provided with a polymer coating 142 corresponding to that of the top roll 13.

[0019] The intermediate rolls 15—22 in the stack of rolls 12 consist alternately of hard-faced and soft-faced rolls in such a way that, except the middle nip N_5 of the stack of rolls 12, in all the other nips one of the rolls is a hard-faced roll and the roll forming a nip therewith is a soft-faced roll. By soft-faced calender rolls are meant both modern rolls provided with a thin polymer coating and rolls provided with a coating made of a sheet material by pressing in an axial direction and conventionally used in the supercalender. The middle nip N_5 in the stack of rolls 12 is a so-called soft nip, in which both of the rolls forming the nip are soft-faced rolls. Thus, as the top roll 13 in the stack of rolls is a soft-faced roll, the uppermost intermediate roll 15 is a hard-faced roll. In a corresponding way, the second topmost intermediate roll 16 is a soft-faced roll provided with a resilient polymer coating 166 and, further, the following intermediate roll 17, or the third intermediate roll from the top, is a hard-face roll. As already stated above, the rolls 18 and 19 forming the middle nip N_5 are soft rolls provided with a resilient polymer coating 186,196, the next roll 20 downwards being again a hard-faced roll. The second lowest intermediate roll 21 is a roll provided with a polymer coating 216 and the lowermost intermediate roll 22 is a hard-faced roll.

[0020] In the calender shown in Fig. 1, bearing housings 151,161,171,181,191,201,211,221 of each intermediate roll 15—22 in the stack of rolls 12 are mounted on arms 152,162, 172,182,192,202,212,222, which are linked pivotally on the calender frame 11 by means of articulated joints 153,163,173,183,193,203,213 and 223 parallel to the axis of said rolls. The arms 152,162,172,182,192,202,212 and 222 are provided

with relief devices 155, 165,175,185,195,205,215 and 225, which are hydraulic cylinders, which are attached at one end thereof to said arms of the intermediate rolls 15—22 and at the opposite end thereof to the calender frame 11. The so-called journal loads of the intermediate rolls, ie the loads caused by the bearing housings of the intermediate rolls 15—22 and by the auxiliary devices attached to the bearing housings, can be relieved by means of said relief devices. The required relief forces are of different magnitude on the different intermediate rolls 15—22 since, as already stated previously, the take-out rolls, which are attached to the bearing housings of the intermediate rolls 15—22, are situated alternately on the inside and on the outside of the nip plane relative to the calender frame 11, and thus the loading caused by them on the relief devices is of different magnitude.

[0021] It has also already been stated above that in a situation where the stack of rolls is quickly opened the rolls 13—22 in the stack of rolls must be detached from one another in such a way that each of the nips N_1 — N_9 opens, preferably an equal distance. In Fig. 1, which thus shows the calender 10 in a running position with the nips N_1 — N_9 closed, attempts are made to illustrate this in such a way that the length of the relief device 155 of the topmost intermediate roll 15, ie the length of a projecting piston rod of a hydraulic cylinder, is denoted with the reference sign a. The size of the gap between the rolls required in quick-opening is in turn denoted with the reference sign b in Fig. 1. Consequently, the length of the projecting piston rod in the hydraulic relief device 165 of the second highest intermediate roll 16 must be a + b.

[0022] In a similar way, in the direction downwards in the stack of rolls 12, the length of the projecting piston rod in the relief device of each intermediate roll must be by said distance b greater than the length of the piston rod in the relief device situated above so that in the calender 10 shown in Fig. 1 comprising eight intermediate rolls the length of the projecting piston rod in the hydraulic relief device 225 of the lowermost intermediate roll 22 is a + 7b. In that case, when in a quick-opening situation the stack of rolls is opened by means of the lower cylinder 143 by lowering the bottom roll 14, there remains a gap of the size of the distance b in each nip N_1 — N_9 when the piston rods of the hydraulic relief devices are at the bottom. Naturally, a gap larger than this may remain in the lowermost nip N_9 depending on the extent of the movement of the lower cylinder 143. Thus, during quick-opening the roll under each intermediate roll falls by the distance b more than the roll above. Because of the stresses directed by the quick-opening at the hydraulic relief devices 155,165,175,185,195,205,215 and 225, cushioned hydraulic cylinders must be used as relief devices. The operation of the system during quick-opening is described more closely by means of the hydraulic circuit diagram shown in Fig. 2.

[0023] Fig. 2 thus shows a hydraulic circuit diagram which accomplishes the method and the system in accordance with the invention. As already previously clearly stated, the invention primarily relates to the quick-opening of a set of rolls in a calender, and the invention is accomplished in such a way that the internal flow resistance of the hydraulic system is utilized during the quick-opening. The system thus enables the separation of the rolls of the system of rolls from one another as quickly as possible after a quick-opening signal has been received. In Fig. 2, the hydraulic relief devices of the intermediate rolls, ie the relief cylinders, are denoted with the reference signs 155,165,175,185,195,205,215 and 225 corresponding to Fig. 1. The diagram drawing shown in Fig. 2 also depicts the lowermost intermediate roll 22 of the set of rolls, which roll is the drive roll of the calender, and the articulated arm 222 of said drive roll, by which it is attached with an articulated joint to the calender frame not shown in Fig. 2.

[0024] The relief cylinders 155—225 of each intermediate roll in the set of rolls have a valve of their own 156,166,176,186,196,206,216 and 226, through which a required relief pressure is supplied to the relief cylinders in a running situation, and through which valves 156—226 the pressure is discharged from the relief cylinders 155—225 in a situation where the calender is quick-opened. In accordance with Fig. 2, said valves 156—226 are preferably hydraulically controlled valves, for instance, directional control valves shown in the figure. The hydraulic circuit diagram of the system is divided, in a way, into four sections such that, firstly, the hydraulic system comprises a first pressure circuit 31', through which the same filling pressure is supplied to the side of the piston rod of all the relief cylinders 155—225. Thus, the relief cylinders are double-acting hydraulic cylinders, as shown by Fig. 2. The filling pressure is supplied to the side of the piston rod of the relief cylinders mainly for the reason that, in a quick-opening situation, the relief cylinders can be made shorter quickly by the action of said filling pressure when their relief pressure has been discharged.

[0025] The first pressure circuit comprises an adjustable valve 31, for instance, a controlled pressure valve, through which pressure is supplied from a pressure supply line of the system to a pressure line 31b leading to the side of the piston rod of the relief cylinders 155—225. The first pressure circuit is additionally provided with a pressure gauge 31a or equivalent, by which the pressure of the first pressure circuit is monitored. Further, the first pressure circuit is fitted with a reservoir 30 for a pressure medium, or hydraulic oil, said reservoir being preferably placed at the upper part of the hydraulic system. The purpose of the reservoir 30 is to ensure a sufficiently rapid supply of oil to the side of the rod of the relief cylinders 155—225 in a quick-opening situation.

[0026] The hydraulic system comprises, in addition, separate pressure circuits 32',33',34' for the relief cylinders 165,185,205 of the intermediate rolls provided with

an external take-out roll, for the relief cylinders 155,175,195,215 of the intermediate rolls provided with an internal take-out roll and separately for the relief cylinder 225 of the lowermost intermediate roll, ie the drive roll 22. These separate pressure circuits are necessary specifically for the reason that, on one hand, the different locations of the take-out rolls produce different loads on the relief cylinders, so naturally it must be possible to load them with a different pressure. On the other hand, the loads directed to the drive roll 22 differ from the load of all the other intermediate rolls, so said drive roll 22 must have a separate pressure circuit of its own. Thus, each of these separate pressure circuits 32',33',34' comprises an adjustable valve 32,33,34, which is preferably of a type corresponding to the valve 31 of the first pressure circuit 31', through which valve hydraulic oil from the pressure supply line of the hydraulic system is passed to respective pressure lines 32b,33b,34b. Further, these pressure circuits are provided with pressure gauges 32a,33a,34a or equivalent means. Hydraulic oil is passed from said pressure circuits to the relief cylinders 155—225 through the valves 156—226. As already previously stated, said valves 156—226 are hydraulically controlled valves and these valves receive their necessary control pressure through the valve 35, which thus controls jointly all the valves 156—226 of the relief cylinders 155—225.

[0027] Fig. 2 specifically shows a quick-opening situation where the pressures have been discharged under the piston of the relief cylinders 155—225. In such a quick-opening situation, pressure has been removed from the loading cylinder 143 of the bottom roll 14 of the calender, in which case the bottom roll 14 has been lowered. Before the quick-opening situation, the hydraulic system has been in an operation state corresponding to Fig. 2A, where the valve 35 controlling the valves 156—226 of the relief cylinders 155—225 has supplied the hydraulically controlled valves 156—226 with a control pressure, which has maintained said valves against the spring-load of the valves in the position shown in Fig. 2A in such a way that the hydraulic oil has been allowed to get under the piston of the relief cylinders 155—225 through the pressure circuits 32',33',34' in order to provide a required relief force. In the quick-opening situation shown in Fig. 2, said valve 35 has moved to the position shown by said Fig. 2, in which case the control pressure of the valves 156—226 of the relief cylinders 155—225 has discharged and the spring-load of the valves has displaced the valves from the position shown in Fig. 2A to the position shown in Fig. 2, in which the pressure connection from the pressure circuits 32',33',34' to the relief cylinders 155—225 has been cut off and, correspondingly, the connection from below the piston of the relief cylinders 155—225 to a reservoir 40 has been opened.

[0028] As clearly depicted in Fig. 2, in a quick-opening situation the valves 156—226 of the relief cylinders 155—225 are connected in series in such a way that

the hydraulic oil flowing out from each relief cylinder 155—225 is guided through the valve 166—226 of the next relief cylinder below, in which case, in the fashion shown by Fig. 2, the hydraulic oil flowing out from all the relief cylinders 155—225 finally passes through the valve 226 of the relief cylinder 225 of the lowermost intermediate roll, ie the drive roll 22. This means that the hydraulic oil flowing out from the relief cylinder 155 of the uppermost intermediate roll 15 meets the highest flow resistance, because the oil flowing out from said relief cylinder has to pass through the valves of the relief cylinders of all the intermediate rolls 15—22. The flow resistance decreases correspondingly in the direction downwards in the set of rolls.

[0029] As shown in Fig. 1, in a quick-opening situation the lowest intermediate roll, or the drive roll 22, must perform the largest vertical movement, while the required movement of the topmost intermediate roll 15 is the smallest. In accordance with the illustration of Fig. 1, the necessary displacement of the intermediate rolls increases in the direction from above downwards. This in turn means that, if it is desired to cause all the nips $N_1—N_9$ of the set of rolls to open simultaneously, the opening or falling velocity of the lowest intermediate roll 22 must be the highest and this falling velocity must decrease in the direction upwards in the set of rolls. In the invention this condition is satisfied because, as already stated above, the rate of the flow out from the relief cylinder 225 of the lowermost intermediate roll 22 is the highest, since the flow resistance is correspondingly the lowest. The flow resistance increases in the direction upwards, in which case the rate of flow of the oil out from the relief cylinders naturally decreases. This, of course, affects linearly the movement velocity of the relief cylinders. Thus, in the arrangement in accordance with the invention, gaps begin to be formed between the rolls 13—22 immediately after the valves 156—226 of the relief cylinders 155—225 have moved from the position shown in Fig. 2A to the position shown in Fig. 2.

[0030] As the hydraulic drawing sign representing the valve 35 in Figs. 2 and 2A seeks to illustrate, an electromagnetically controlled valve is preferably used as the valve 35. Besides providing a very quick action, said valve 35 can be easily connected electrically in the electric system of the calender 10, for instance, to such an automatic system that identifies web breaks, if such an automatic system is available in the calender.

[0031] The above description of the invention has described, with reference to the figures, an arrangement in which the flow resistances in the path of the hydraulic oil flowing out from the relief cylinders 155—225 are arranged in such a way that the calender nips $N_1—N_9$ open simultaneously in the entire roll stack 12. It is, however, completely possible to arrange and to regulate the flow resistances in the path of the hydraulic oil flowing out from the relief cylinders 155—225 in such a way that the nips $N_1—N_9$ can be caused to open in a controlled way in the order considered to be necessary. However,

as stated, the hydraulic circuit described provides the simultaneous opening of the nips.

[0032] It is also clear from the above description that in a quick-opening situation the intermediate rolls 5 15—22 of the set of rolls have to be supported by the relief cylinders 155—225 in their entirety. Thus, in particular in a quick-opening situation, the relief cylinders are subjected to quite considerable loads, so cylinders provided with cushioning shall be used as the relief cylinders. 10 In spite of this, and especially for the reason that the invention provides a very fast opening of the nips $N_1—N_9$, it may be necessary to also use a separate stop which is based, for instance, on friction and which receives totally or at least partly the loads arising from quick-opening.

[0033] As already explained above, it is advantageous to cause the roll nips to open simultaneously in the quick-opening of the set of rolls in a supercalender. In a certain way this might be considered problematic because, as it is stated clearly, for instance, in connection with Fig. 1, the opening distances of the roll nips $N_1—N_9$ of the set of rolls are unequal in length and because of this the opening speeds of the nips must also be different. It must be possible to take care of this variation of opening distances and opening speeds by means of the hydraulic system and, in particular, by means of the relief cylinders connected to the hydraulic system. Figs. 3A—3C, 4A—4C and 5A—5C illustrate some alternative solutions as to how the relief cylinders 15 20 can be accomplished in a manner fulfilling the requirements set on them.

[0034] Figs. 3A—3C show a hydraulic cylinder, which is generally denoted with the reference numeral 50. When conventional hydraulic cylinders comprise, as a rule, two operating positions, for instance, a pressurized working position and an unpressurized free position, the hydraulic cylinder 50 comprises, in addition to these two positions, also a third operating position, which is between these two positions and to which position the hydraulic cylinder can be stopped without complicated control circuits. The hydraulic cylinder 50 comprises a cylinder part 51, inside which a piston 52 is fitted in a normal way so as to be axially movable. The bottom of the cylinder part 51 is replaced by a partition wall 55, through which a special measuring cylinder 53 is connected to the cylinder part 51. A second piston 54, or the piston of the measuring cylinder, is fitted inside the measuring cylinder 53 so as to be also axially movable. Thus, in the arrangement shown in Figs. 3A—3C the measuring cylinder 53 functions directly as an extension of the cylinder part 51. The partition wall 55 is provided with a large-diameter opening extending therethrough, the extension 56 of the piston of the measuring cylinder being capable of being pushed from the measuring cylinder 53 into the cylinder part 51. Thus, end faces 58,59 of the piston 52 and the piston 54 of the measuring cylinder can be brought against each other so as to be in contact with each other, as illustrated in Fig. 3A. A spring 25 30 35 40 45 50 55 60 65 70 75 80 85

means 57, for instance, a set consisting of cup springs, a spiral spring or equivalent is fitted to the bottom of the measuring cylinder 53, said spring serving as cushioning for the piston 54 of the measuring cylinder. The measuring cylinder 53 is provided with a pressure connection 60 and with a drain connection 61 and the cylinder part 51 of the hydraulic cylinder is provided with a pressure connection 62 and with a drain connection 63, respectively. The control pressure 64 of the hydraulic cylinder 50 is passed to these pressure connections 60,62 in such a way that the control pressure is passed to the pressure connection 62 of the cylinder part 51 through valves 66,67, which in the illustration of Figs. 3A—3C are one-way restrictor valves allowing free flow in one direction and restricting it in the other direction (said directions are opposite in the valves 66,67). Similarly, the drain connections 61,63 are connected to a drain duct 65.

[0035] In Fig. 3A the hydraulic cylinder 50 is shown in an unpressurized state, in which case the piston 52 is pressed to the bottom of the cylinder part 51 by the action of the external load acting on the piston 52. When applied, for instance, to the calender shown in Fig. 1, the situation of Fig. 3A may correspond to a service position, in which case all pressures have been removed from the relief cylinders. As shown by Fig. 3A, the piston 52 is pressed against the piston 54 of the measuring cylinder further in such a way that the piston 54 of the measuring cylinder is pressed against the spring means 57 situated at the bottom of the measuring cylinder 53.

[0036] Fig. 3B shows the hydraulic cylinder 50 in its first working position, which, when applied to the calender of Fig. 1, corresponds to the running position of the calender, in which the calendering nips are closed. In that case, oil has been supplied from the pressure duct 64 through the pressure connection 60 to the measuring cylinder 53 and through the pressure connection 62 to the cylinder part 51. The piston 54 of the measuring cylinder has then moved together with the piston 52 in the illustration of Fig. 3B to the right, or outwards from the cylinder, until the piston 54 of the measuring cylinder has stopped at the partition wall 55. Before resting on the partition wall 55, the piston extension 56 of the measuring cylinder has rested against the piston 52 in such a way that the ends 58,59 of the pistons are against each other to balance pressure forces. This is due to the fact that the same pressure prevails on both sides of the piston 54 of the measuring cylinder and that the area on the side of the partition wall 55 of the piston 54 of the measuring cylinder is smaller. When the pressure has been further increased, hydraulic oil has flown between the pistons 54 and 52 through the pressure connection 62, until the piston 52 has reached its extreme position, that is, until an external obstruction has stopped the movement of the piston 52. In the calender this means precisely the situation shown in Fig. 1, where the nips are closed. Fig. 3C shows a situation where pressure has been removed from the pressure duct 64. When ap-

plied to the relief cylinders of the calender, this corresponds to a quick-opening situation. In that case, when pressure has been removed from the duct 64, the piston 54 of the measuring cylinder is immediately pressed to the bottom of the measuring cylinder 53 against the spring means 57 by the action of an external load. The spring means 57 thus provides cushioning in the hydraulic cylinder. The piston 52 of the cylinder part 51 has then naturally moved at the same speed an equivalent distance to the left in Fig. 3C and, if the valve 66 is controlled so as to be closed, the distance between the pistons 54 and 52 remains constant and the piston 52 stops at the position shown in Fig. 3C. Pressure is allowed to discharge freely through the valve 67, since said valve 67 restricts the flow and the pressure only in one direction. If again the valve 66 is not controlled so as to be closed, oil discharges from the cylinder part 51 at a desired speed through said valve 66, until the situation corresponding to Fig. 3A is reached. If the valve 66 is replaced, for instance, by a pressure-controlled lock valve, it is possible to move in both directions between the positions shown in Figs. 3B and 3C depending on the position of said valve 66 and on the pressure prevailing in the pressure duct 64. The extent of the movement of the piston rod of the piston 52 in the cylinder part 51 between the positions shown in Figs. 3B and 3C is determined by the play of the piston 54 of the measuring cylinder 53 and by the diameters of the cylinders.

[0037] In the construction of Figs. 3A—3C the measuring cylinder 53 is placed directly in connection with the cylinder part 51, but it can also be placed apart from the cylinder part, since with regard to operation a mechanical connection between the piston 54 of the measuring cylinder and the piston 52 of the cylinder part is not necessary. Such arrangements have been illustrated by way of example in Figs. 4A—4C and 5A—5C. The numbering of said figures corresponds to the numbering of Figs. 3A—3C in so far as the constructions correspond to each other. In Figs. 4A—4C, however, the hydraulic cylinder has been denoted generally with the reference numeral 50a and in Figs. 5A—5C with the reference numeral 50b.

[0038] The hydraulic cylinder 50a shown in Figs. 4A—4C largely corresponds in structure to the structure shown in Figs. 3A—3C such that also in the arrangement of Figs. 4A—4C a measuring cylinder 53 is placed directly as an extension of a cylinder part 51. In the structure shown in Figs. 4A—4C, a partition wall 55, however, cuts off the mechanical connection between a piston 52 of the cylinder part and a cylinder 54 of the measuring cylinder, while a connecting duct 55a is formed into the partition wall 55 so as to extend therethrough so that the cylinder part 51 and the measuring cylinder 53 are in hydraulic connection with each other. A pressure connection 62 opens into said connecting duct 55a. The operation of the hydraulic cylinder 50a is equivalent to that already explained in connection with Figs. 3A—3C.

[0039] In the position depicted in Fig. 4A, the hydraulic

cylinder 50a is unpressurized. An external force acting on the arm of the piston 52 in the cylinder part 51 presses the piston 52 against the partition wall 55 and, in a similar way, a spring means 57 fitted to the bottom of the measuring cylinder 53 presses the piston 54 of the measuring cylinder against the partition wall 55. When hydraulic oil is supplied from a pressure duct 64 into the hydraulic cylinder through pressure connections 60 and 62, the piston 54 of the measuring cylinder remains in the position shown in Fig. 4B against the partition wall 55 and the piston 52 of the cylinder part 51 moves to the position of Fig. 4B. The piston 54 of the measuring cylinder remains in place for the reason that the same pressure prevails on both sides of said piston and that the effective area below the piston 54 of the measuring cylinder is considerably larger than on the side of the partition wall 55. Thus, the hydraulic cylinder 50a is shown in Fig. 4B in a position corresponding to that of Fig. 3B and in this regard reference is made to the description in connection with Fig. 3B.

[0040] In Fig. 4C the hydraulic cylinder 50a is shown in a situation corresponding to that of Fig. 3C. In that case the control pressure prevailing in the pressure duct 64 has been removed, and the pressure below the piston 52 of the cylinder part 51 and prevailing in the connecting duct 55a has pressed the piston 54 of the measuring cylinder to the bottom of the measuring cylinder. In other respects, especially regarding the operation of valves 66, 67, reference is made to the description already completely given in connection with Fig. 3C. If the pressure is discharged totally out from the cylinder part 51, the piston 52 is first pressed to the bottom of the cylinder part 51 against the partition wall 55 and, after that, the spring means 57 pushes the piston 54 of the measuring cylinder from the other side against the partition wall 55. This, then, is the situation shown in Fig. 4A.

[0041] Figs. 5A—5C depict a further embodiment of a hydraulic cylinder 50b, which is in operation identical with the arrangement shown in Figs. 4A—4C. Also in structure, the arrangement of Figs. 5A—5C differs from that of Figs. 4A—4C only in the respect that the measuring cylinder 53 in this embodiment is not directly an axial extension of the cylinder part 51, but, instead, the measuring cylinder 53 is turned so as to be at an angle relative to the cylinder part 51, said angle being 90° in Figs. 5A—5C. Regarding the operation of this application, reference is made completely to that which has already been explained in connection with Figs. 3A—3C and 4A—4C.

[0042] The movements of the hydraulic cylinder illustrated in Figs. 3A—3C, 4A—4C and 5A—5C can also be easily provided with different movement velocities. A desired return movement from a working position may be determined for the cylinder. In operation, this kind of cylinder is suitable especially for a place where the length of the cylinder in working position varies from time to time and it is, however, desired that the piston of the cylinder shall move specifically a desired distance in a

direction away from the working position. This kind of operation is needed, in addition to the relief cylinders of the calender in accordance with the invention, in a working cylinder of a brake, where the length of the working position of the cylinder changes with the wearing down of the brake material and, nevertheless, it is possible to provide, after the braking has stopped, a clearance of constant magnitude for the brake in order to prevent dragging. This property has been achieved by passing the oil discharging from the cylinder during a return movement to a special measuring cylinder. The length of the return movement of the cylinder can be affected in a desired manner by means of the diameter and the length of the measuring cylinder.

[0043] Above, the invention has been described by way of example with reference to the figures in the accompanying drawing. The invention is, however, not confined to the exemplifying embodiments shown in the figures alone, but different embodiments of the invention may vary within the scope of the inventive idea defined in the accompanying claims.

Claims

1. A method of quick-opening a set of rolls in a calender, in particular a supercalender, in which calender a paper or board web (W) to be calendered is passed through calendering nips (N₁—N₉) formed by a variable-crown top roll (13), a variable-crown bottom roll (14) and by two or more intermediate rolls (15—22) fitted between the top and bottom rolls (13, 14), which rolls (13—22) are arranged to form a substantially vertical stack of rolls (12), and in which calender the intermediate rolls (15—22) are supported by means of hydraulic relief cylinders (155—225) so as to relieve the nip load produced by the masses of bearing housings (151—221) of said intermediate rolls (15—22) and of auxiliary devices (154—224) associated therewith, whereby during the quick-opening of the set of rolls, the bottom roll (14) of the set of rolls is lowered and the relief pressures in the hydraulic relief cylinders (155—225) are discharged completely or partly so as to open the calendering nips (N₁—N₉), and whereby the hydraulic oil flowing out from the relief cylinders (155—225) is controlled by means of valves (156—226) connected to the relief cylinders (155—225) of each intermediate roll (15—22) so that, during quick-opening of the set of rolls, the roll nips (N₁—N₉) may be opened in a controlled way in a desired order.
2. A method as claimed in claim 1, characterized in that, in connection with quick-opening of the set of rolls, the valves (156—226) controlling the relief cylinders (155—225) of all the intermediate rolls (15—22) are connected to open substantially si-

multaneously.

3. A method as claimed in claim 1 or 2, **characterized in that**, during quick-opening, the valves (156—226) controlling the relief cylinders (155—225) are controlled in such a way that, in the flow path of the oil flowing out from the relief cylinders (155—225), the oil flowing out from the relief cylinder (155) of the uppermost intermediate roll (15) has the highest flow resistance, the flow resistance being lower in the oil flowing out from the relief cylinder of each intermediate roll below in such a way that the oil flowing out from the relief cylinder (225) of the lowermost intermediate roll (22) has the lowest flow resistance, in which case all the calendering nips (N₁—N₉) open substantially simultaneously. 5

4. A method as claimed in any one of the preceding claims, **characterized in that**, in a situation where the set of rolls is opened quickly, the hydraulic oil flowing out from the relief cylinders (155—225) is passed through the valve controlling the next relief cylinder below, in which case the pressure of the relief cylinder (155) of the uppermost intermediate roll (15) is discharged through all the valves (156—226) controlling the relief cylinders, the pressure of the relief cylinder (225) of the lowermost intermediate roll (22) being discharged only through its own valve (226), respectively. 10

5. A method as claimed in any one of the preceding claims, **characterized in that** the valves (156—225) of the relief cylinders (155—225) of the intermediate rolls (15—22) are controlled hydraulically by a valve (35), which provides a control impulse simultaneously to all the valves (156—226) of the relief cylinders (155—225). 15

6. A method as claimed in any one of the preceding claims, **characterized in that**, during quick-opening of the set of rolls, the movement of the piston (52) of the relief cylinders (50,50a,50b) is stopped or the movement is at least highly decelerated before the piston (52) is moved to the bottom after a desired quick-opening distance has been reached. 20

7. A method as claimed in claim 6, **characterized in that** the movement of the piston (52) of the relief cylinders (50,50a,50b) is controlled by means of a measuring cylinder (53) connected to a cylinder part (51). 25

8. A calender, in particular a supercalender, having a set of rolls comprising a variable-crown top roll (13), a variable-crown bottom roll (14) and two or more intermediate rolls (15—22) fitted between the top roll (13) and the bottom roll (14), which rolls are arranged to form a substantially vertical stack of rolls (12), the rolls (13—22) placed one above the other in the set of rolls being in nip contact with one another and forming calendering nips (N₁—N₉) therebetween, in which set of rolls the intermediate rolls (15—22) are provided with hydraulic relief cylinders (155—225) so as to relieve the nip load produced by the masses of bearing housings (151—221) of the intermediate rolls (15—22) and of auxiliary devices associated therewith, such as, take-out rolls (154—224), the calender comprising a hydraulic system for the set of rolls which is arranged to supply hydraulic oil to the relief cylinders (155—225) under a desired pressure during operation of the calender, which pressure depends on the load acting on the relief cylinders (155—225), **characterized in that** the hydraulic system is arranged to supply hydraulic oil to the relief cylinders (155—225) through valves (156—226) and that, during quick-opening of the set of rolls when the bottom roll (14) in the stack of rolls (12) has been lowered, hydraulic pressure is discharged from the relief cylinders (155—225) in a controlled way by means of said valves (156—226) so that the calendering nips (N₁—N₉) may be opened in a desired order. 30

9. A calender as claimed in claim 8, **characterized in that** the hydraulic system is divided into several separate pressure circuits in such a way that at least the hydraulic relief cylinders (155,175,195,215) of the intermediate rolls (15,17,19,21) provided with a take-out roll (154,174,194,214) situated on the inside relative to a calender frame (11) have a pressure circuit (33') of their own, the hydraulic relief cylinders (165,185,205) of the intermediate rolls (16,18,20) provided with a take-out roll (164,184,204) situated on the outside relative to the calender frame (11) have a pressure circuit (32') of their own, the relief cylinder (225) of the drive roll (22) of the calender has a pressure circuit (34') of its own and, in addition, the hydraulic system comprises a separate pressure circuit (31'), which is arranged to supply each relief cylinder (155—225) with a counterpressure of equal magnitude acting in a direction opposite to the relief pressure. 35

10. A calender as claimed in claim 8 or 9, **characterized in that**, in a situation where the set of rolls is opened quickly, the valves (156—226) controlling the relief cylinders (155—225) are arranged to discharge the relief pressure acting in the relief cylinders (155—225) substantially simultaneously. 40

11. A calender as claimed in any one of claim 8—10, **characterized in that** the flow resistances in the flow paths of the hydraulic oil flowing out from the relief cylinders (155—225) are arranged in such a way that the flow resistance in the flow path of the 45

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uppermost relief cylinder (155) is the highest and the flow resistance in the flow path of the lowermost relief cylinder (225) is the lowest, the flow resistance being stepwise reduced in the direction from the relief cylinder (155) of the uppermost intermediate roll (15) to the relief cylinder (225) of the lowermost intermediate roll (22).

12. A calender as claimed in any one of claims 8—11, **characterized in that**, in a situation where the set of rolls is opened quickly, the valves (156—226) controlling the relief cylinders (155—225) are connected in series in such a way that the oil flowing out from the relief cylinders (155—225) passes through the valve of the next relief cylinder below, in which case the oil flowing out from the uppermost relief cylinder (155) passes through all the valves (156—226) controlling the relief cylinders and the pressure of the lowermost relief cylinder (225) discharges only through its own valve (226), respectively.

13. A calender as claimed in any one of claims 8—12, **characterized in that** the valves (156—226) of the relief cylinders (155—225) are hydraulically controlled valves and that the hydraulic system is provided with a valve (35), which is arranged to supply a control pressure to all the valves (156—226) of the relief cylinders.

14. A calender as claimed in any one of claims 8—13, **characterized in that** the hydraulic system is provided an oil reservoir (30), which is connected to the pressure circuit (31') supplying a counterpressure to the relief cylinders (155—225), in order to ensure a sufficiently quick supply of oil to the cylinders in a quick-opening situation.

15. A calender as claimed in any one of claims 8—14, **characterized in that** the relief cylinders (50,50a, 50b) are three-position hydraulic cylinders, the movement of whose piston (52) can be stopped or at least highly decelerated during quick-opening of the set of rolls after a desired quick-opening distance has been reached before the piston (52) is moved to the bottom.

16. A calender as claimed in claim 15, **characterized in that** a measuring cylinder (53) controlling the movement of the piston (52) of the relief cylinder during quick-opening is connected to the relief cylinders (50,50a,50b).

17. A calender as claimed in claim 16, **characterized in that** the coupling between the measuring cylinder (53) and the relief cylinder (50) is such that the pistons (52,54) of the measuring and relief cylinders can be brought to a mechanical connection with each other.

18. A calender as claimed in claim 16, **characterized in that** the pistons (52,54) of the relief cylinder (50a, 50b) and the measuring cylinder (53) are in direct hydraulic connection with each other.

Patentansprüche

1. Verfahren zum schnellen Öffnen eines Satzes an Walzen bei einem Kalander insbesondere bei einem Superkalander, wobei dem Kalander eine zu kalandrierende Papierbahn oder Pappbahn (W) durch Kalanderspalten (N_1 - N_9) tritt, die durch eine obere Walze (13) mit variabler Bombierung, einer unteren Walze (14) mit variabler Bombierung und durch zwei oder mehrere Zwischenwalzen (15 - 22) ausgebildet sind, die zwischen der oberen und der unteren Walze (13, 14) eingesetzt sind, wobei die Walzen (13 - 22) so angeordnet sind, dass sie einen im Wesentlichen vertikalen Stapel an Walzen (12) ausbilden, und wobei in dem Kalander die Zwischenwalzen (15 - 22) mittels Hydraulikentlastungszylin dern (155 - 225) so gestützt sind, dass sie die durch die Massen von Lagergehäusen (151 - 221) der Zwischenwalzen (15 - 22) und von zu diesen zugehörigen Hilfsvorrichtungen (154 - 224) erzeugte Spaltlast entlasten, wodurch während des schnellen Öffnens des Satzes an Walzen die untere Walze (14) des Satzes an Walzen abgesenkt wird und die Freigabedrücke bei den Hydraulikentlastungszylin dern (155 - 225) vollständig oder teilweise abgegeben werden, um so die Kalanderspalte (N_1 - N_9) zu öffnen und wodurch das von den Entlastungszylin dern (155 - 225) herausströmende Hydrauliköl mittels Ventilen (156 - 226), die mit den Entlastungszylin dern (155 - 225) von jeder Zwischenwalze (15 - 22) verbunden sind, so gesteuert wird, dass während des schnellen Öffnens des Satzes an Walzen die Walzenspalte (N_1 - N_9) in einer gesteuerten Weise in einer erwünschten Reihenfolge geöffnet werden können.
2. Verfahren gemäß Anspruch 1, dadurch gekennzeichnet, dass in Zusammenhang mit dem schnellen Öffnen des Satzes der Walzen die Ventile (156 - 226), die die Entlastungszylin dern (155 - 225) sämtlicher Zwischenwalzen (15 - 22) steuern, verbunden sind, um im Wesentlichen gleichzeitig zu öffnen.
3. Verfahren gemäß Anspruch 1 oder 2, dadurch gekennzeichnet, dass während des schnellen Öffnens die Ventile (156 - 226), die die Entlastungszylin dern (155 - 225) steuern, in einer derartigen Weise gesteuert werden, dass in der Strömungsbahn des aus den Ent-

lastungszylindern (155 - 225) herausströmenden Öles das aus dem Entlastungszylinder (155) der obersten Zwischenwalze (15) herausströmende Öl den höchsten Strömungswiderstand hat, wobei der Strömungswiderstand in dem aus dem Entlastungszylinder von jeder darunter befindlichen Zwischenwalze herausströmende Öl in einer derartigen Weise niedriger ist, dass das aus dem Entlastungszylinder (225) der obersten Zwischenwalze (22) herausströmende Öl den niedrigsten Strömungswiderstand hat, wobei in diesem Fall sämtliche Kalanderspalten (N_1 - N_9) im Wesentlichen gleichzeitig öffnen.

4. Verfahren gemäß einem der vorherigen Ansprüche, dadurch gekennzeichnet, dass

in einer Situation, bei der der Satz an Walzen schnell geöffnet wird, das aus den Entlastungszylindern (155 - 225) herausströmende Hydrauliköl durch das Ventil tritt, das den nächsten darunter befindlichen Entlastungszylinder steuert, wobei in diesem Fall der Druck des Entlastungszylinders (155) der obersten Zwischenwalze (15) über sämtliche Ventile (155 - 226), die die Entlastungszylinder steuern, abgegeben wird, wobei der Druck des Entlastungszylinders (225) der untersten Zwischenwalze (22) lediglich durch sein eigenes Ventil (226) jeweils abgegeben wird.

5. Verfahren gemäß einem der vorherigen Ansprüche, dadurch gekennzeichnet, dass

die Ventile (156 - 225) der Entlastungszylinder (155 - 225) der Zwischenwalzen (15 - 22) hydraulisch durch ein Ventil (35) gesteuert werden, das einen Steuerimpuls gleichzeitig für sämtliche Ventile (156 - 226) der Entlastungszylinder (155 - 225) vorsieht.

6. Verfahren gemäß einem der vorherigen Ansprüche, dadurch gekennzeichnet, dass

während des schnellen Öffnens des Satzes an Walzen die Bewegung des Kolbens (52) der Entlastungszylinder (50, 50a, 50b) angehalten wird oder die Bewegung zumindest außerordentlich verlangsamt wird, bevor der Kolben (52) zu dem Boden bewegt wird, nachdem ein erwünschter Schnellöffnungsabstand erreicht worden ist.

7. Verfahren gemäß Anspruch 6, dadurch gekennzeichnet, dass

die Bewegung des Kolbens (52) der Entlastungszylinder (50, 50a, 50b) mittels eines Messzylinders (53) gesteuert wird, der mit einem Zylinderabschnitt (51) verbunden ist.

8. Kalander insbesondere ein Superkalander mit einem Satz an Walzen, der eine obere Walze (13) mit variabler Bombierung, eine untere Walze (14) mit

variabler Bombierung und zwei oder mehr Zwischenwalzen (15 - 22) aufweist, die zwischen der oberen Walze (13) und der unteren Walze (14) eingesetzt sind, wobei die Walzen so angeordnet sind, dass sie einen im Wesentlichen vertikalen Stapel an Walzen (12) ausbilden, wobei die in dem Satz an Walzen übereinander angeordneten Walzen (13 - 22) in einem Spaltkontakt miteinander stehen und zwischen ihnen Kalanderspalten (N_1 - N_9) ausbilden, wobei in dem Satz an Walzen die Zwischenwalzen (15 - 22) mit Hydraulikentlastungszylindern (155 - 225) versehen sind, um die durch die Massen von Lagergehäusen (151 - 221) der Zwischenwalzen (15 - 22) und von zu diesen zugehörigen Hilfsvorrichtungen wie beispielsweise Herausnehmwalzen (154 - 224) erzeugte Spaltlast zu entlasten, wobei der Kalander ein Hydrauliksystem für den Satz an Walzen aufweist, dass so eingerichtet ist, dass es Hydrauliköl zu den Entlastungszylindern (155 - 225) bei einem erwünschten Druck während des Betriebs des Kalanders liefert, wobei dieser Druck von der an den Entlastungszylindern (155 - 225) wirkenden Last abhängig ist, dadurch gekennzeichnet, dass

das Hydrauliksystem so eingerichtet ist, dass es Hydrauliköl zu den Entlastungszylindern (155 - 225) über Ventile (156 - 226) liefert und während des schnellen Öffnens des Satzes an Walzen, wenn die untere Walze (14) bei dem Stapel an Walzen (12) abgesenkt worden ist, der Hydraulikdruck von den Entlastungszylindern (155 - 225) in einer gesteuerten Weise mittels der Ventile (156 - 226) abgegeben wird, so dass die Kalanderspalte (N_1 - N_9) in einer gewünschten Reihenfolge geöffnet werden können.

9. Kalander gemäß Anspruch 8, dadurch gekennzeichnet, dass

das Hydrauliksystem in mehrere separate Druckleitungen in einer derartigen Weise geteilt ist, dass zumindest die Hydraulikentlastungszylinder (155, 175, 195, 205) der Zwischenwalzen (15, 17, 19, 21), die mit einer Herausnehmwalze (154, 174, 194, 214) versehen sind, die sich an der Innenseite relativ zu einem Kalanderrahmen (11) befindet, eine Druckleitung (33') für sich allein haben, wobei die Hydraulikentlastungszylinder (165, 185, 205) der Zwischenwalzen (16, 18, 20), die mit einer Herausnehmwalze (164, 184, 204) versehen sind, die sich an der Außenseite relativ zu dem Kalanderrahmen (11) befindet, eine Druckleitung (32') für sich allein haben, wobei der Entlastungszylinder (225) der Antriebswalze (22) von dem Kalander eine Druckleitung (34') für sich allein hat und außerdem das Hydrauliksystem eine separate Druckleitung (31') aufweist, die so eingerichtet ist, dass sie jeden Entla-

stungszyliner (155 - 225) mit einem Gegendruck mit einer gleichen Größe beliefert, der in einer zu dem Freigabedruck entgegengesetzten Richtung wirkt.

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10. Kalander gemäß Anspruch 8 oder 9, dadurch gekennzeichnet, dass in einer Situation, bei der der Satz an Walzen schnell geöffnet wird, die die Entlastungszyliner (155 - 225) steuern den Ventile (156 - 226) so eingerichtet sind, dass sie den in den Entlastungszylinern (155 - 225) wirkenden Freigabedruck im Wesentlichen gleichzeitig abgeben.

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11. Kalander gemäß einem der Ansprüche 8 bis 10, dadurch gekennzeichnet, dass die Strömungswiderstände in den Strömungsbahnen des aus den Entlastungszylinern (155 - 225) herausströmenden Hydrauliköls in einer derartigen Weise eingerichtet sind, dass der Strömungswiderstand in der Strömungsbahn des obersten Entlastungszyliners (155) am höchsten ist und dass der Strömungswiderstand in der Strömungsbahn des untersten Entlastungszyliners (225) am geringsten ist, wobei der Strömungswiderstand in der Richtung von dem Entlastungszyliner (155) der obersten Zwischenwalze (15) zu dem Entlastungszyliner (225) der untersten Zwischenwalze (22) schrittweise verringert ist.

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12. Kalander gemäß einem der Ansprüche 8 bis 11, dadurch gekennzeichnet, dass in einer Situation, bei der der Satz an Walzen schnell geöffnet wird, die die Entlastungszyliner (155 - 225) steuern den Ventile (156 - 226) in Reihe in einer derartigen Weise verbunden sind, dass das aus den Entlastungszylinern (155 - 225) herausströmende Öl durch das Ventil des nächsten darunter befindlichen Entlastungszyliners tritt, wobei in diesem Fall das aus dem obersten Entlastungszyliner (155) herausströmende Öl durch sämtliche Ventile (156 - 226) tritt, die die Entlastungszyliner steuern, und der Druck des untersten Entlastungszyliners (225) lediglich durch sein eigenes Ventil (26) abgegeben wird.

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13. Kalander gemäß einem der Ansprüche 8 bis 12, dadurch gekennzeichnet, dass die Ventile (156 bis 226) der Entlastungszylinrer (155 - 225) hydraulisch gesteuerte Ventile sind und das Hydrauliksystem mit einem Ventil (35) versehen ist, das eingerichtet ist, um einen Steuerdruck zu sämtlichen Ventilen (156 - 226) der Entlastungszylinrer zu liefern.

14. Kalander gemäß einem der Ansprüche 8 bis 13,

dadurch gekennzeichnet, dass das Hydrauliksystem mit einem Ölspeicher (30) versehen ist, der mit der Druckleitung (31') verbunden ist, die einen Gegendruck zu den Entlastungszylinern (155 - 225) liefert, um ein ausreichend schnelles Liefern des Öles zu den Zylindern bei einer Schnellöffnungssituation sicherzustellen.

15. Kalander gemäß einem der Ansprüche 8 bis 14, dadurch gekennzeichnet, dass die Entlastungszylinrer (50, 50a, 50b) Dreipositions-Hydraulikzylinrer sind, wobei die Bewegung ihres Kolbens (52) während des schnellen Öffnens des Satzes an Walzen angehalten werden kann oder zumindest außerordentlich verzögert werden kann, nachdem ein erwünschter Schnellöffnungsabstand erreicht worden ist, bevor der Kolben (52) zu dem Boden bewegt worden ist.

16. Kalander gemäß Anspruch 15, dadurch gekennzeichnet, dass ein Messzyliner (53), der die Bewegung des Kolbens (52) des Entlastungszyliners während des schnellen Öffnens steuert, mit den Entlastungszylinern (50, 50a, 50b) verbunden ist.

17. Kalander gemäß Anspruch 16, dadurch gekennzeichnet, dass das Kuppeln zwischen dem Messzyliner (53) und dem Entlastungszyliner (50) derart ist, dass die Kolben (52, 54) des Messzyliners und des Entlastungszyliners miteinander in eine mechanische Verbindung gebracht werden können.

18. Kalander gemäß Anspruch 16, dadurch gekennzeichnet, dass die Kolben (52, 54) der Entlastungszylinrer (50a, 50b) und des Messzyliners (53) miteinander in einer direkten hydraulischen Verbindung stehen.

Revendications

1. Procédé pour l'ouverture rapide des rouleaux d'un ensemble de rouleaux d'une calandre, en particulier une supercalandre, calandre dans laquelle une bande (W) de papier ou de carton à calandrer est amenée à passer par des lignes de contact de calandrage (N₁ - N₉) formées par un rouleau supérieur (13) à bombé variable, un rouleau inférieur (14) à bombé variable et par au moins deux rouleaux intermédiaires (15 - 22) installés entre les rouleaux supérieur et inférieur (13, 14), lesquels rouleaux (13 à 22) sont agencés pour former un empilement de rouleaux (12) sensiblement vertical, et calandre dans laquelle les rouleaux intermédiaires (15 à 22) sont supportés à l'aide de cylindres hydrauliques de relâchement (155 - 225) afin de relâcher la charge

de pincement créée par les masses des corps (151 - 221) de paliers desdits rouleaux intermédiaires (15 - 22) et de dispositifs auxiliaires (154 - 224) associés à ceux-ci, grâce à quoi, pendant l'ouverture rapide de l'ensemble de rouleaux, le rouleau inférieur (14) de l'ensemble de rouleaux s'abaisse et les pressions de relâchement dans les cylindres hydrauliques de relâchement (155 - 225) sont relâchées entièrement ou partiellement de façon à ouvrir les lignes de contact de calandrage (N_1 - N_9), et grâce à quoi l'huile hydraulique sortant des cylindres de relâchement (155 - 225) est commandée à l'aide de soupapes (156 - 226) connectées aux cylindres de relâchement (155 - 225) de chaque rouleau intermédiaire (15 - 22) de façon que, pendant l'ouverture rapide de l'ensemble de rouleaux, les lignes de contact (N_1 - N_9) des rouleaux puissent s'ouvrir d'une manière commandée dans un ordre voulu.

2. Procédé selon la revendication 1, caractérisé en ce que, en liaison avec l'ouverture rapide de l'ensemble de rouleaux, les soupapes (156 - 226) commandant les cylindres de relâchement (155 - 225) de tous les rouleaux intermédiaires (15 - 22) sont couplées de façon à s'ouvrir sensiblement en même temps.

3. Procédé selon la revendication 1 ou 2, caractérisé en ce que, pendant l'ouverture rapide, les soupapes (156 - 226) commandant les cylindres de relâchement (155 - 225) sont commandées de façon que, sur le trajet d'écoulement de l'huile sortant des cylindres de relâchement (155 - 225), l'huile sortant du cylindre de relâchement (155) du rouleau intermédiaire (15) situé le plus haut présente la plus grande résistance à l'écoulement, la résistance à l'écoulement étant moindre dans l'huile sortant du cylindre de relâchement de chaque rouleau intermédiaire situé au-dessous, de telle sorte que l'huile sortant du cylindre de relâchement (225) du rouleau intermédiaire (22) le plus bas présente la plus faible résistance à l'écoulement, auquel cas toutes les lignes de contact de calandrage (N_1 - N_9) s'ouvrent de manière sensible simultanée.

4. Procédé selon l'une quelconque des revendications précédentes, caractérisé en ce que, dans une situation où l'ensemble de rouleaux s'ouvre rapidement, l'huile hydraulique sortant des cylindres de relâchement (155 - 225) est amenée à passer par la soupape commandant le cylindre de relâchement suivant situé au-dessous, auquel cas la pression du cylindre de relâchement (155) du rouleau intermédiaire (15) situé le plus haut est relâchée par l'intermédiaire de toutes les soupapes (156 - 226) commandant les cylindres de relâchement, la pression du cylindre de relâchement (225) du rouleau inter-

5. Procédé selon l'une quelconque des revendications précédentes, caractérisé en ce que les soupapes (156 - 226) des cylindres de relâchement (155 - 225) des rouleaux intermédiaires (15 - 22) sont commandées de manière hydraulique par une soupape (35) qui fournit simultanément une impulsion de commande à toutes les soupapes (156 - 226) des cylindres de relâchement (155 - 225).

6. Procédé selon l'une quelconque des revendications précédentes, caractérisé en ce que, pendant l'ouverture rapide de l'ensemble de rouleaux, le mouvement du piston (52) des cylindres de relâchement (50, 50a, 50b) est interrompu ou le mouvement est au moins fortement ralenti avant que le piston (52) ne descende jusqu'au fond après qu'une distance d'ouverture rapide voulue a été atteinte.

7. Procédé selon la revendication 6, caractérisé en ce que le mouvement du piston (52) des cylindres de relâchement (50, 50a, 50b) est commandé à l'aide d'un cylindre de mesure (53) couplé à une pièce (51) de cylindre.

8. Calandre, en particulier supercalandre, ayant un ensemble de rouleaux comprenant un rouleau supérieur (13) à bombé variable, un rouleau inférieur (14) à bombé variable et au moins deux rouleaux intermédiaires (15 - 22) installés entre le rouleau supérieur (13) et le rouleau inférieur (14), lesquels rouleaux sont agencés pour former un empilement de rouleaux (12) sensiblement vertical, les rouleaux (13 - 22) superposés dans l'ensemble de rouleaux étant en contact à pincement les uns avec les autres et formant entre eux des lignes de contact de calandrage (N_1 - N_9), ensemble de rouleaux dans lequel les rouleaux intermédiaires (15 - 22) sont munis de cylindres hydrauliques de relâchement (155 - 225) servant à relâcher la charge de pincement produite par les masses de logements (151 - 221) de paliers des rouleaux intermédiaires (15 - 22) et de dispositifs auxiliaires associés à ceux-ci, par exemple des rouleaux extracteurs (154 - 224), la calandre comportant pour l'ensemble de rouleaux un système hydraulique agencé pour fournir de l'huile hydraulique aux cylindres de relâchement (155 - 225) sous une pression voulue pendant le fonctionnement de la calandre, laquelle pression dépend de la charge agissant sur les cylindres de relâchement (155 - 225), caractérisée en ce que le système hydraulique est agencé pour fournir de l'huile hydraulique aux cylindres de relâchement (155 - 225) par l'intermédiaire de soupapes (156 - 226) et en ce que, pendant l'ouverture rapide de

l'ensemble de rouleaux, lorsque le rouleau (14) du bas de la pile de rouleaux (12) a été abaissé, la pression hydraulique est relâchée dans les cylindres de relâchement (155 - 225) d'une manière commandée par l'intermédiaire desdites soupapes (156 - 226), de façon que les lignes de contact de calandrage (N_1 - N_9) puissent s'ouvrir dans un ordre voulu.

9. Calandre selon la revendication 8, caractérisée en ce que le système hydraulique est divisé en plusieurs circuits de pression séparés, de façon qu'au moins les cylindres hydrauliques de relâchement (155, 175, 195, 215) des rouleaux intermédiaires (15, 17, 19, 21) pourvus d'un rouleau extracteur (154, 174, 194, 214) situé à l'intérieur par rapport à un bâti (11) de calandre aient leur propre circuit de pression (33'), que les cylindres hydrauliques de relâchement (165, 185, 205) des rouleaux intermédiaires (16, 18, 20) pourvus d'un rouleau extracteur (164, 184, 204) situé à l'extérieur par rapport au bâti (11) de calandre aient leur propre circuit de pression (32'), que le cylindre de relâchement (225) du rouleau d'entraînement (22) de la calandre ait son propre circuit de pression (34') et, en outre, le système hydraulique comporte un circuit de pression séparé (31') conçu pour fournir à chaque cylindre de relâchement (155 - 225) une contre-pression de valeur égale agissant dans une direction opposée à la pression de relâchement.

10. Calandre selon la revendication 8 ou 9, caractérisée en ce que, dans le cas où l'ensemble de rouleaux s'ouvre rapidement, les soupapes (156 - 226) commandant les cylindres de relâchement (155 - 225) sont agencées pour décharger de manière sensiblement simultanée, la pression de relâchement agissant dans les cylindres de relâchement (155 - 225).

11. Calandre selon l'une quelconque des revendications 8 à 10, caractérisée en ce que la résistance à l'écoulement dans les circuits d'écoulement de l'huile hydraulique sortant des cylindres de relâchement (155 - 225) est conçue de façon que la résistance à l'écoulement dans le trajet d'écoulement du cylindre de relâchement (155) situé le plus haut soit maximale et que la résistance à l'écoulement sur le trajet d'écoulement du cylindre de relâchement (225) le plus bas soit minimale, la résistance à l'écoulement étant réduite par paliers depuis le cylindre de relâchement (155) du rouleau intermédiaire (15) le plus haut jusqu'au cylindre de relâchement (225) du rouleau intermédiaire (22) le plus bas.

12. Calandre selon l'une quelconque des revendications 8 à 11, caractérisée en ce que, dans le cas où l'ensemble de rouleaux s'ouvre rapidement, les soupapes (156 - 226) commandant les cylindres de relâchement (155 - 225) sont montées en série de façon que l'huile sortant des cylindres de relâchement (155 - 225) passe par la soupape du cylindre de relâchement suivant situé au-dessous, auquel cas l'huile qui sort du cylindre de relâchement (155) situé le plus haut passe par toutes les soupapes (156 - 226) commandant les cylindres de relâchement et la pression du cylindre de relâchement (225) le plus bas ne se décharge respectivement que par sa propre soupape (226).

13. Calandre selon l'une quelconque des revendications 8 à 12, caractérisée en ce que les soupapes (156 - 226) des cylindres de relâchement (155 - 225) sont des soupapes à commande hydraulique et en ce que le système hydraulique comporte une soupape (35) conçue pour fournir une pression de commande à toutes les soupapes (156 - 226) des cylindres de relâchement.

14. Calandre selon l'une quelconque des revendications 8 à 13, caractérisée en ce que le système hydraulique comporte un réservoir (30) d'huile relié au circuit de pression (31') fournissant une contre-pression aux cylindres de relâchement (155-225) afin d'assurer une alimentation en huile suffisamment rapide des cylindres dans le cas d'une ouverture rapide.

15. Calandre selon l'une quelconque des revendications 8 à 14, caractérisée en ce que les cylindres de relâchement (50, 50a, 50b) sont des cylindres hydrauliques à trois positions, le mouvement du piston (52) de ces cylindres pouvant être arrêté ou au moins fortement ralenti pendant l'ouverture rapide des rouleaux de l'ensemble de rouleaux après qu'une distance d'ouverture rapide voulue a été atteinte avant que le piston (52) ne parvienne au fond.

16. Calandre selon la revendication 15, caractérisée en ce qu'un cylindre de mesure (53) commandant le mouvement du piston (52) du cylindre de relâchement pendant une ouverture rapide est couplé avec les cylindres de relâchement (50, 50a, 50b).

17. Calandre selon la revendication (16), caractérisée en ce que l'accouplement entre le cylindre de mesure (53) et le cylindre de relâchement (50) est tel que les pistons (52, 54) des cylindres de mesure et de relâchement peuvent se trouver accouplés mécaniquement les uns avec les autres.

18. Calandre selon la revendication 16, caractérisée en ce que les pistons (52, 54) du cylindre de relâchement (50a, 50b) et le cylindre de mesure (53) sont accouplés les uns avec les autres par une

liaison hydraulique directe.

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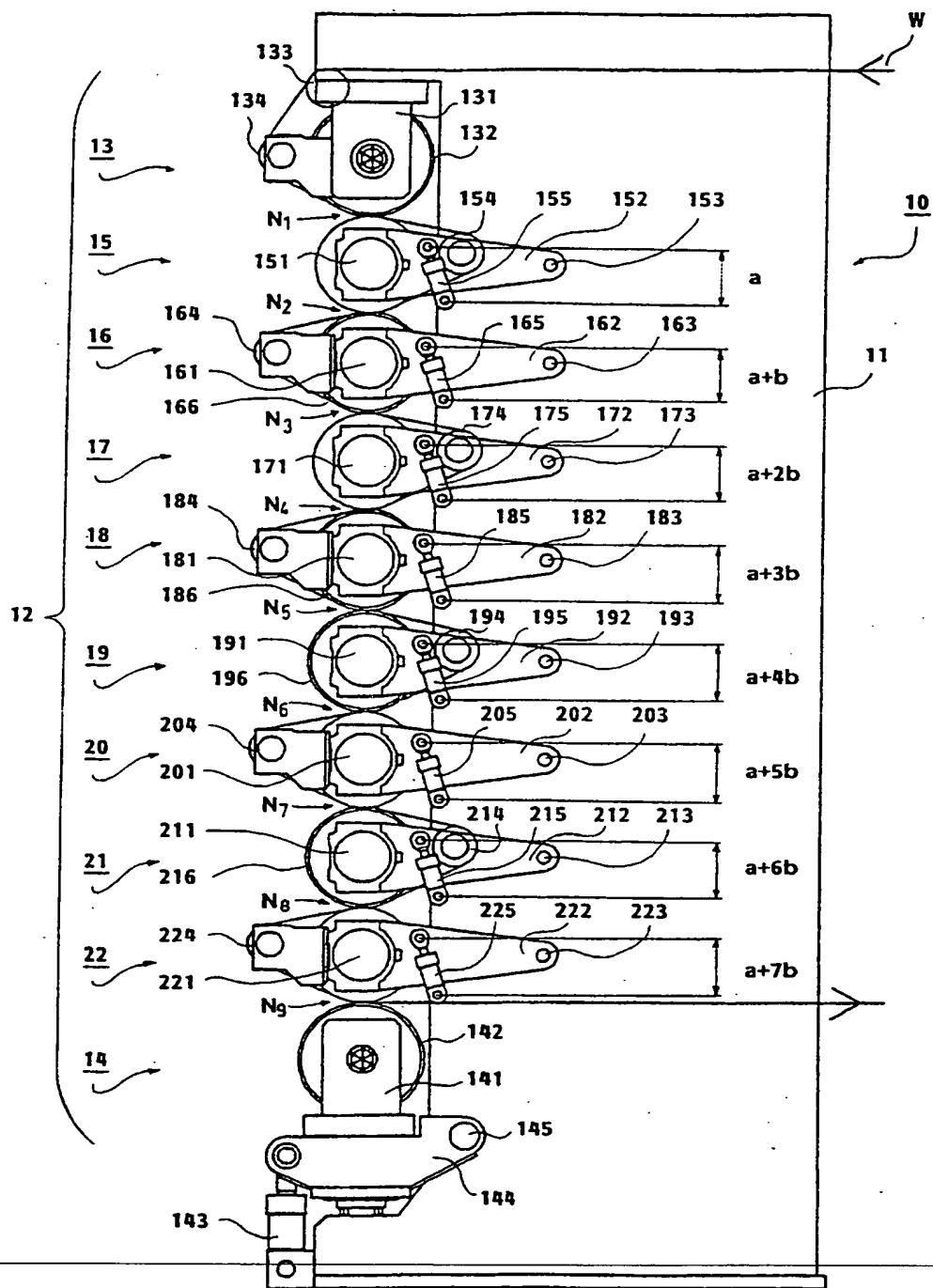


FIG. 1

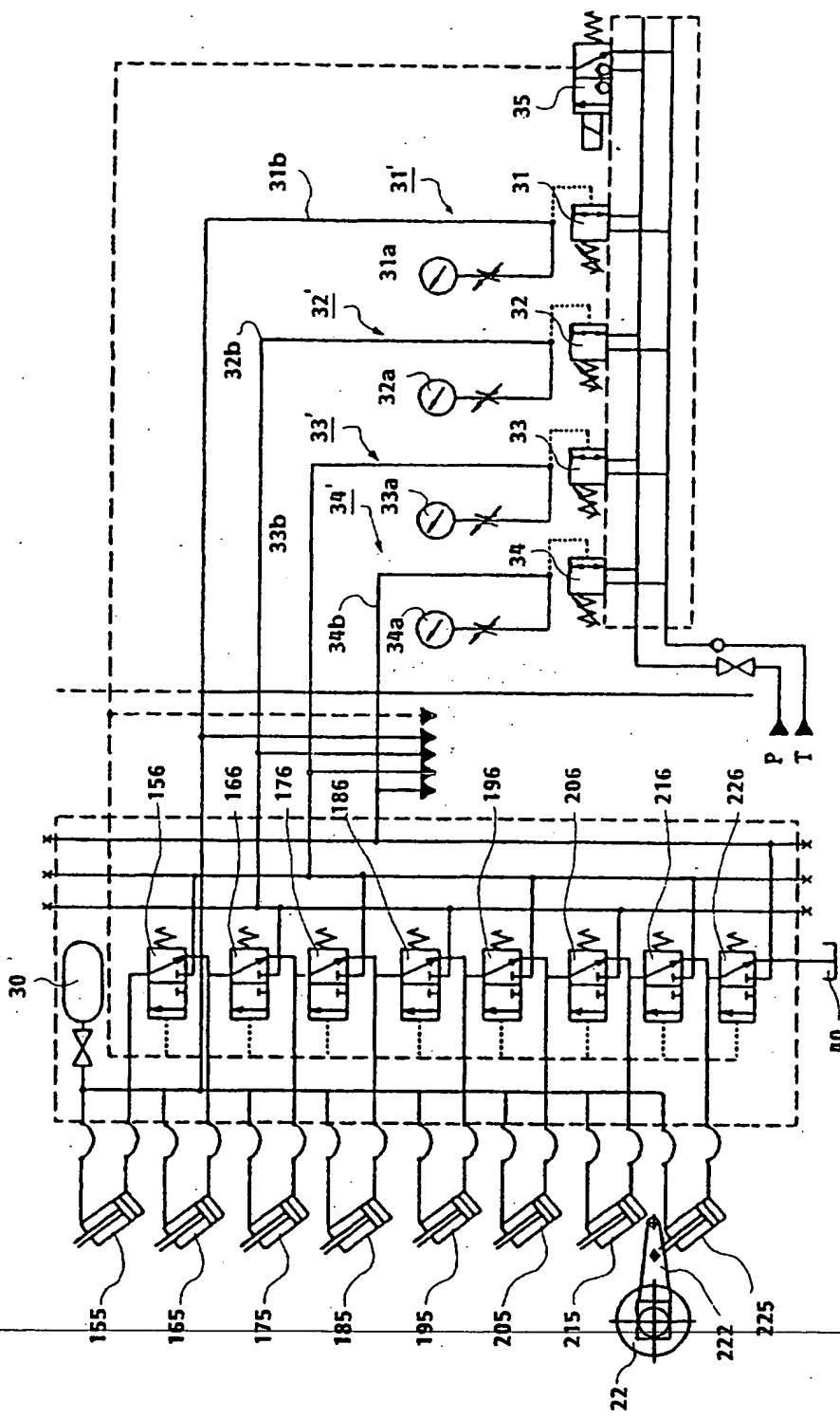


FIG. 2

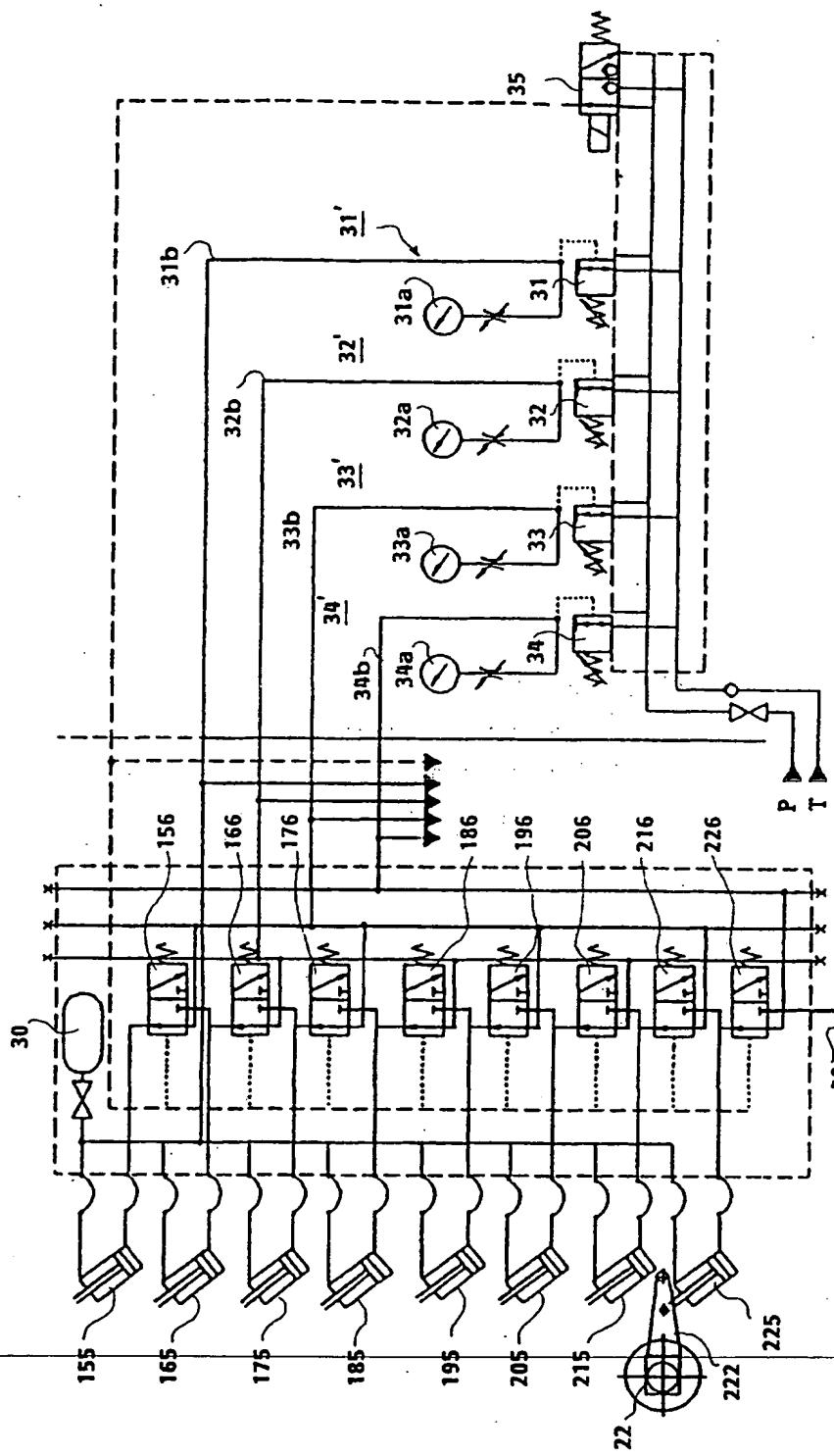


FIG. 2A

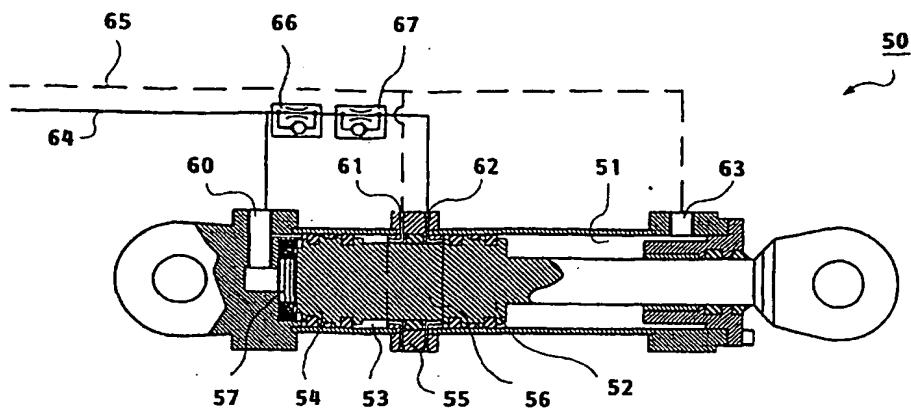


FIG. 3A

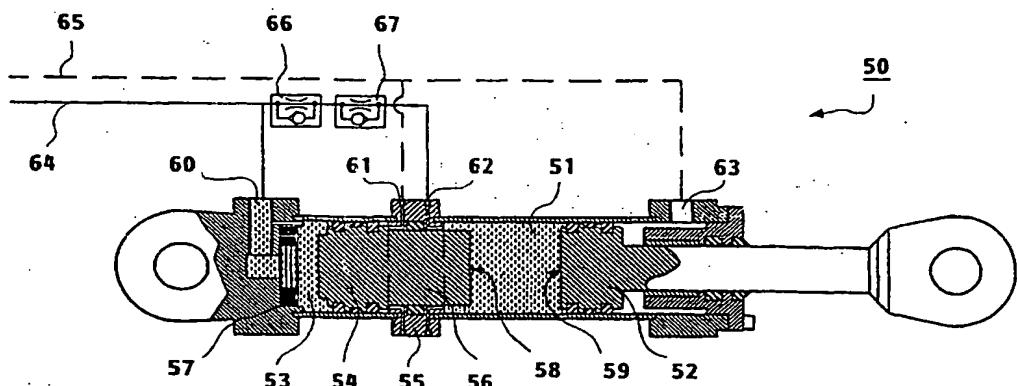


FIG. 3B

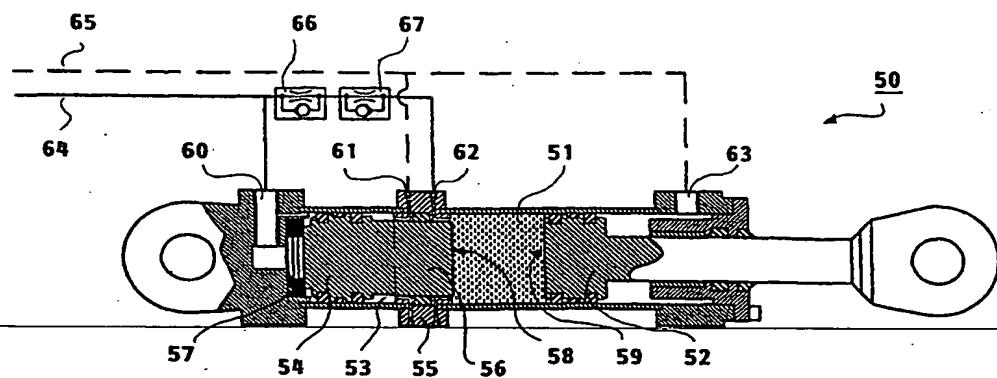


FIG. 3C

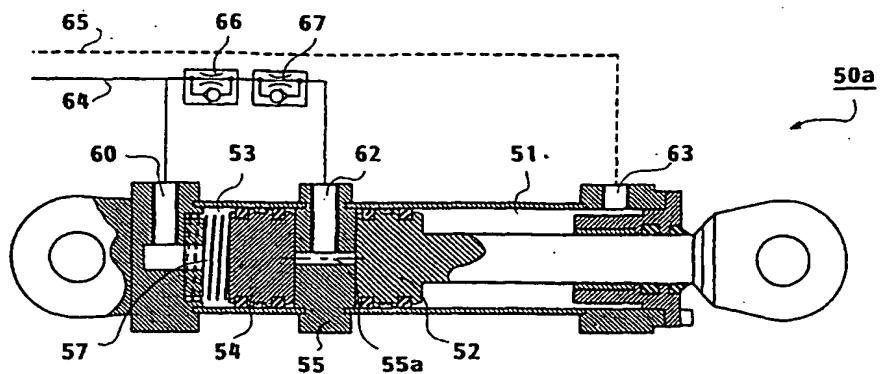


FIG. 4A

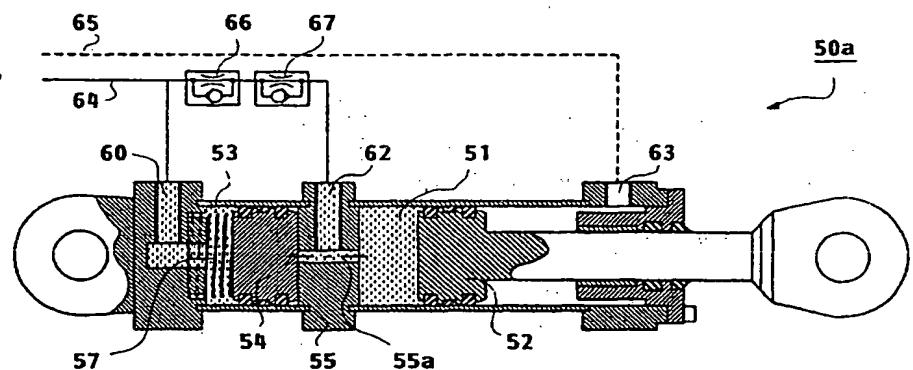


FIG. 4B

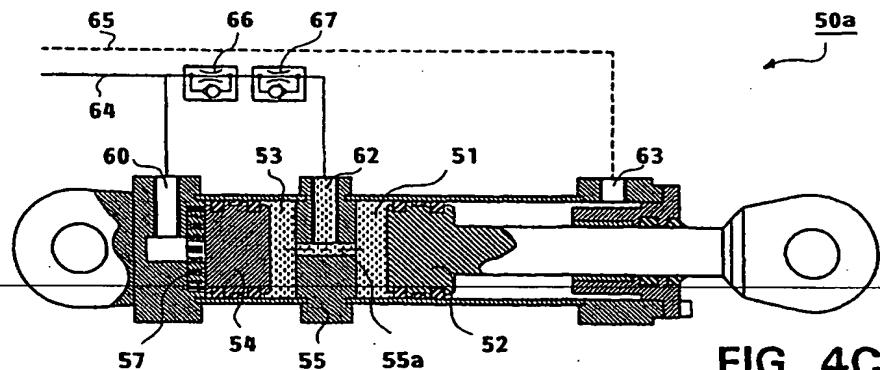


FIG. 4C

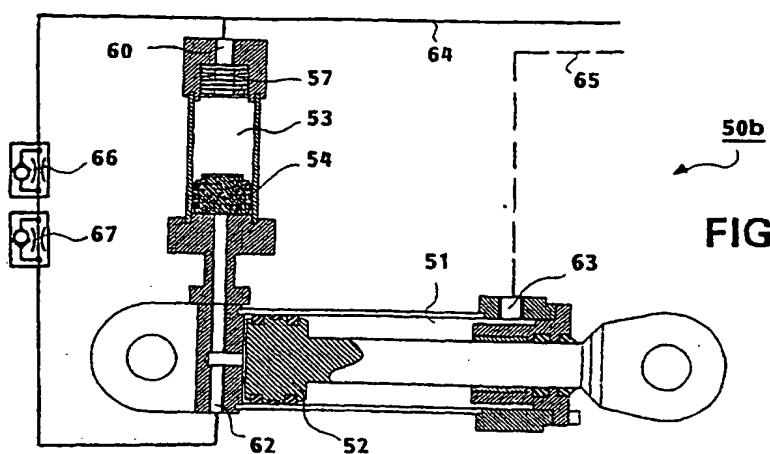


FIG. 5A

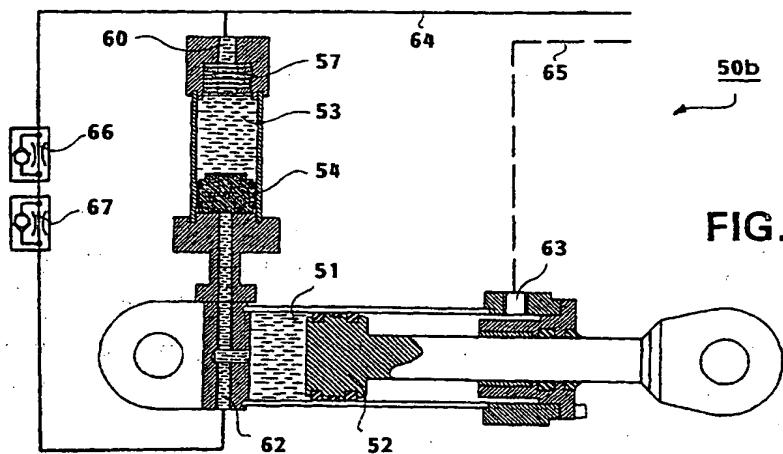


FIG. 5B

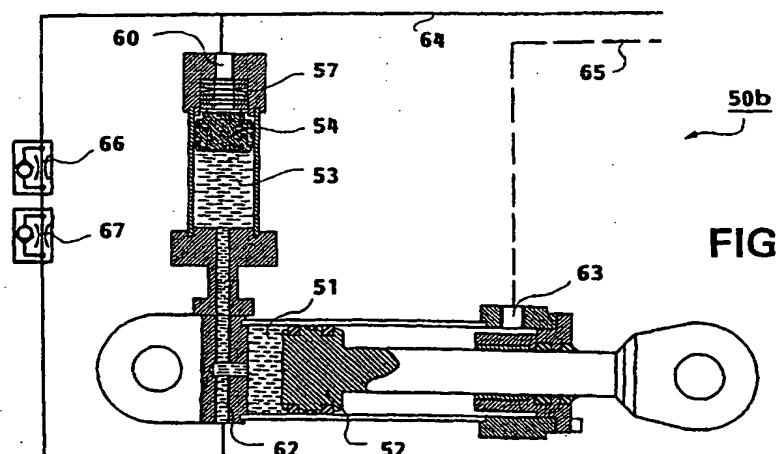


FIG. 5C